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TEST REPORT FOR DRILL STRING SEAL PRESSURE TEST

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1	/	Safety JA Harvey 2/1/96 S7-07				MJ Schliebe L6-13				3	-
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3	-	cc: TJ Kelley S7-21									
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Test Report for Drill String Seal Pressure Test

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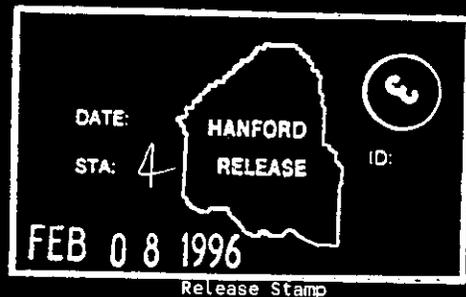
Abstract:

A basic question was asked concerning the drill string which is used in rotary mode coring operations: "...what is the volume leak rate loss in a drill rod string under varying conditions of the joint boxes and pins being either dry or coated with lubricant...". A variation of this was to either have an o-ring installed or absent on the drill rod that was grooved on the pin. A series of tests were run with both the o-ring grooved Longyear drill rod and the plain pin end rod manufactured by Diamond Drill. Test results show that drill rod leakage of both types is lowered dramatically when thread lubricant is applied to the threaded joints and the joints made up tight. The Diamond Drill rod with no o-ring groove has virtually no leakage when used with thread lubricant and the joints are properly tightened.

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TEST REPORT
FOR
DRILL STRING
SEAL PRESSURE TEST

PREPARED BY

JIM F. MCCORMICK

Characterization Equipment Engineering
Tank Waste Remediation System

Westinghouse Hanford Company
Richland, Washington
January 1996

Table of Contents

1.0	Introduction	3
2.0	Test Equipment and Method	3
2.1	Test Equipment	3
2.2	Test Method	4
2.2.1	Longyear	4
2.2.2	Diamond Drill	4
3.0	Test Results	5
3.1	Longyear O-Ring Pipe	5
3.2	Diamond Drill Pipe	6
4.0	Conclusions and Recommendations	6
4.1	Conclusions	6
4.2	Recommendations	6
5.0	Appendix A - Data Tables and Calculations	8
6.0	Appendix B - Data Graphics	9
7.0	Appendix C - Photos	10

Drill String Seal Pressure Test

1.0 Introduction

The basic problem was: "...to determine the volume leak rate loss in a drill rod string under varying conditions of the joint boxes and pins being either dry or with lubricant...". A variation of this was to either have an o-ring installed or absent on the drill rod that was grooved on the pin.

In order to provide data on the leak rate of a typical drill string joint seal assembly, used with rotary coring work, a series of tests were performed in the 305 Building, Engineering Test Laboratory (ETL). The original test plan (WHC-SD-WM-TP-427) was for the Longyear o-ring type drill rod that is typically used for rotary drilling. Additional tests were performed using "Diamond Drill" rod without the o-ring grooves. Nitrogen gas is normally used during the sampling effort to pressurize the drill string during the rotary coring process and sampler changes. Nitrogen also provides bit cooling and removal of waste cuttings from the bottom of the core hole. Testing was started and finished during January 1996.

2.0 Test Equipment and Method

Laboratory air was used in place of nitrogen for the testing efforts. The test consists of: 1) screwing the box and pin threaded connections together on several joints of drill rod, 2) installing a blanking flange on one end of the string and a pressure inlet adapter on the other end, 3) pressurizing the length of drill string to approximately 100 psig, and 4) measuring the elapsed time between 10 psig pressure drops down to a minimum of 10 psig or a total elapsed time of 10 minutes, whichever comes first.

2.1 Test Equipment

The Longyear drill rod dimensions are 2.25-inch o.d. by 1.916-inch i.d. pipe with a 0.167-inch wall thickness. An o-ring groove is cut near the base of the pin end for an extra sealing surface when a rubber o-ring is installed in the groove. The test string consisted of three 1-foot, one 2-foot, one 5-foot and one 19-inch sections of pipe used in the 141 3/8-inch test length. Sealing of the threaded joints is by thread interference fit, o-ring rubber seal, lubricant on the threads and joint makeup.

The Diamond Drill rod consisted of 2.25-inch o.d. by 1.916-inch i.d. pipe with a 0.167-inch wall thickness. There is no o-ring groove at the base of the pin thread connection. Sealing of the string is by thread interference fit, lubricant on the threads and joint makeup. This test string consisted of four 1-foot, one 2-foot, one 6-inch, one 5-inch and

one 19-inch section with an o-ring groove from the Longyear test string. Length of the string without the o-ring groove joint was 142 5/8-inches. This test string arrangement was 143 5/8-inches long when the 19-inch o-ring joint replaced one 1-foot joint and the 6-inch joint.

Other equipment used was a small sub with a blanking plate welded on the end and an air pressure inlet adapter consisting of an old test bit, tees, nipples, shut off valve, pressure gauge, stop watch and thread lubricants. An Ashcroft temperature compensated and calibrated test gauge (SN 002-31-04-044) was used for all tests. See photos in Appendix C.

2.2 Test Method

2.2.1 Longyear

Three arrangements of the test were run with the Longyear grooved drill rod. Each variation was repeated three times.

- A.) The drill string was made up dry (no thread lubricant) and with no pin end rubber o-rings installed. Refer to Table 1.
- B.) The drill string was made up with no pin end rubber o-rings installed, but lubricant was put on the joint pin threads. Refer to Table 2.
- C.) The drill string was made up with the pin o-rings installed and lubricant was put on the pin threads. New o-rings were installed after each pressure run. Refer to Table 3.

The drill string joints were disconnected after each pressure run, the joints selected at random and made up again for the next pressure run. The third and last run of each test sequence was the beginning joint placement of the next table variation. Jet-Lube thread lubricant was used on these joints.

2.2.2 Diamond Drill

Four arrangements of the test were run with the Diamond Drill rod. Each variation was repeated three times.

- A.) The drill string was made up dry (no thread lubricant).
- B.) The drill string was made up dry with no thread lubricant. One 19-inch o-ring type joint was included and 18 inches (2 joints) of plain pin pipe was removed.

- C.) The drill string in item B) above had Dow D-111 thread lubricant applied to the pin threads.
- D.) The drill string was tested with no o-ring type joint in the string; the two joints (18 inches) of plain pin pipe were added to the test string. All pins were lubricated with Dow D-111.

The drill string joints were disconnected after each pressure run, the joints selected at random and made up again for the next pressure run. The third and last run of each test sequence was the beginning joint placement of the next table variation.

The pressure test in variation D) above was run only once. This test was essentially the fourth sequence of C) with the o-ring joint removed during the fourth pressure run.

All pressure tests required the drill string to be pressured to about 100 psig (which was the laboratory maximum limit). A maximum starting pressure was read and recorded. The shut off valve was closed to trap the air volume and pressure within the test string and the stop watch was started at the time of valve closure. The pressure drop between each 10 psig pressure point was read to a minimum of 10 psig or an elapsed total time of 10 minutes, whichever came first. Volume rate loss was then calculated and used for a graphics representation of the drill string pressure versus volume leak rate. Refer to Appendix A for the hand written copies of the laboratory data tables and calculations. Appendix B contains graphic forms of the data for each arrangement of the drill rod.

3.0 Test Results

3.1 Longyear O-Ring Pipe

A look at Graph 1, Appendix B, shows the assembled dry o-ring type drill rod is leaking quite readily. A moisture check of the joints showed that all joints were leaking.

Graph 2 shows the affect of having a lubricant on the joint threaded connections with no o-ring installed. The flow rate loss dropped from an average above 12.0 cf/hr as shown on Graph 1 to an average below 1.4 cf/hr on Graph 2.

Graph 3 shows virtually no air flow rate or pressure loss when an o-ring is installed and thread lubricant applied.

These graphs show the reading between pressure points and the associated flow rate loss that results during the measured time period.

Note: Caution must be used when the pipe joints are made up with both an o-ring and lubricant on the pin end. If the joint is made up too fast, the lubricant will pile up ahead of the thread interference fit and cause the lubricant to wash the rubber o-ring out of its groove. A potential leak path is thus created.

3.2 Diamond Drill Pipe

Graph 4 shows the dry joint make up of a pipe that has no o-ring groove. Flow rate loss and pressure was very minimal during the 10 minute test period.

Graph 5 shows the leakage that occurs when an o-ring joint is added to the string. All air leakage took place at the o-ring joint box and pin.

Graph 6 shows again that leakage is reduced when lubricant is applied to the drill rod threads and the o-ring joint is left in the test string. All leakage was at the o-ring joint box and pin connections.

Graph 7 without the o-ring joint shows no air leakage during the 10 minute test with thread lubricant applied to the pin ends.

These graphs show the pressure reading between graph points and the associated flow rate loss that results during the measured time period.

4.0 Conclusions and Recommendations

4.1 Conclusions

Test results show that drill rod leakage is lowered dramatically when thread lubricant is applied to the pipe threads and the joints made up tight. This is true even when the o-ring type drill rod (Longyear) has the rubber seal either missing or damaged. The Diamond Drill rod with no o-ring groove has virtually no leakage with thread lubricant and tight joint make up.

4.2 Recommendations

Use the o-ring grooved pipe and get it out of the tank farm supply inventory. Order only non-grooved pipe; it is not necessary to have the extra rubber o-ring for a seal.

Diamond Drill rod can also be purchased cheaper than the o-ring type drill rod.

Omitting the o-ring groove will strengthen the joint box and pin connections during rotary or push sampling operations. This will help eliminate broken drill rod in the tanks, expensive fishing jobs to retrieve broken tools inside the tanks and lost sampling time.

The RMCS crews must pay close attention to their final make up of joints that have o-ring seals. It is possible to wash an o-ring out of its groove with the thread lubricant and thus have a low rate leaking joint.

5.0 Appendix A - Data Tables and Calculations

11/1/96

DRILL STRING PRESSURE TESTS

12 / 2

DATA SHEET: LEAK OFF TEST OF DRILL STRING SEALS

TABLE 1

PRESSURE/TIME (PSIG/SEC)

DRY
NO O-RINGS

1	98 / 0	90 / 4s	80 / 11	70 / 20	60 / 30	50 / 42	40 / 57	30 / 1.15	20 / 1.41	10 / 2.23
2	99 / 0	90 / 3s	80 / 9s	70 / 15	60 / 23	50 / 32	40 / 45	30 / 59	20 / 1.20	10 / 1.53
3	98 / 0	90 / 5s	80 / 15	70 / 29	60 / 44	50 / 1:02	40 / 1:25	30 / 1:53	20 / 2:35	10 / 3:39
4	295 / 0	270 / 12	240 / 35	210 / 64	180 / 97	150 / 136	120 / 197	90 / 247	60 / 336	30 / 475
MI	98.3 / 0	90 / 4s	80 / 11.67	70 / 21.3	60 / 32.3	50 / 45.3	40 / 62.3	30 / 82.3	20 / 112	10 / 158.3

TABLE 2

PINS ONLY — LUBRICANT
NO O-RINGS

1	98 / 0	97 / 10:00	X	/						
2	99 / 0	98 / 10:00	X							
3	98 / 0	90 / 6:22	85 / 10:00	X						
4	295 / 0	285 / 1582	85 / 600							
MI	98.3 / 0	95 / 527.3	85 / 600							

*INST. = INSTANT

JM

1/11/96

DRILL STRING PRESSURE TESTS

2 / 2

DATA SHEET: LEAK OFF TEST OF DRILL STRING SEALS

TABLE 3 PRESSURE/TIME (PSIG/SEC) CHANGE O-RINGS O-RINGS + LUB.

1	99 / 0	98.5 / 10:00	X	/	/	/	/	/	/
2	98 / 0	97 / 10:00	X						
3	100 / 0	99 / 10:00	X						
M	297 / 0	294.5 / 10:00							
MI	99 / 0	98.2 / 10:00							

1									
2									
3									
M									
MI									

*INST. = INSTANT

JM

12
1/14/96

DRILL STRING PRESSURE TESTS

#1
2

DATA SHEET: LEAK OFF TEST OF DRILL STRING SEALS

DIAMOND DRILL ROD

TABLE 4 PRESSURE/TIME (PSIG/SEC) (1) DEY L-142 7/8"

RUN	1	98 /0	91.75 /10:00	X	/	/	/	/	/	/
	2	98 /0	92 /10:00	X						
	3	98 /0	95.5 /10:00	X						
	Σ	98 /0	283.25 /30:00							
	MI	98 /0	92.92 /10:00							

TABLE 7 L-142 7/8" (4) DIAMOND DRILL ROD W/O-111

RUN	1	97.5 /0	97.25 /10:00							
	2									
	3									
	Σ									
	MI									

*INST. = INSTANT

JM

13
1/17/96

DRILL STRING PRESSURE TESTS

2
2

DATA SHEET: LEAK OFF TEST OF DRILL STRING SEALS

L-143 ^{5/8"}

(3) DIAMOND DRILL ROD

TABLE 6

PRESSURE/TIME (PSIE/SEC)

W/O-RING &
D-11

1	98 0	95.5 10:00	X	/	/	/	/	/	NO LEAKS MINOR @ O-RING AND BOX
2	98 0	97 10:00	X						MINOR NO LEAKS @ BLANK SUB @ O-RING JT BOX
3	98 0	97.5 10:00	X						
M	294 0	290 30:00							
MI	98 0	96.57 10:00							

USE C-LEAK ON
JTS TO FIND LEAKS

TABLES

L-143 ^{5/8"}

(2)

DIAMOND DRILL ROD
+ O-RING - DRY

1	98 0	90 4:27	82 10:00	X					O-RING JT LEAK BLANK SUB LEAK
2	99 0	90 2:45	80 6:08	70 1/4 10:00	X				BLANKING SUB LEAK O-RING JT LEAK
3	98 0	90 6:18	81 10:00	X					BLANK SUB LEAK O-RING JT LEAK
M	295 0	270 9:10	243 15:08	70 1/4 10:00					
MI	98.3 0	90 2:33	81 5:22.67	70 1/4 10:00					

*INST. = INSTANT

TEMP. COMPENSATED
ASHCROFT-TEST GAUGE
H2O CALIB.

7-26-96 EXPIRE
4
802-31-04-048
1/4% - 200 PSIG

WHC-SD-WM-TRP-251
REV. 0
Page 8d of 10

JT

357
165
363
1500
388

RUN

1/9/96

DRILL STRING PRESSURE TESTS

JTS: 3-1', 1-2', 1-19", 1-5'
BIT + BLANKING PLATE

LONGYEARTOTAL LENGTH - 141 $\frac{7}{8}$ "

PIPE I.O. - 1.916"

$$V = \frac{\pi d^2}{4} \times L = 0.785 d^2 L$$

$$= 0.785 (1.916)^2 (141.375)$$

$$= 407.4 \text{ IN}^3 \text{ - TEST STRING}$$

$$\approx 408 \text{ IN}^3 \quad \text{" " + INLET CONNECTIONS}$$

$$A = \frac{\pi d^2}{4} = 0.785 d^2$$

$$= 0.785 (1.916)^2$$

$$= 2.882 \text{ IN}^2 \text{ - DRILL PIPE}$$

PROBLEM: FIND THE VOLUME LOSS RATE IN AN
ASSEMBLED DRILL STRING

DIAMOND DRILL

JT: 4-1' 1-2' 1-19" (O-RING)
1-6' 1-5'

BIT + BLANKING PLATE

1) TOTAL LENGTH - 142 $\frac{7}{8}$ " W/O O-RING JT (7 JT)

2) " " " W/O O-RING & OMIT 1-1' & 1-6"

= 143 $\frac{7}{8}$ " (6 JT)VOLUMES

1) DIAMOND DRILL; $V_{DD} = 0.785 (1.916)^2 (142.625)$

$$= 411 \text{ IN}^3 \text{ TEST STRING}$$

$$\approx 411.6 \text{ IN}^3 \quad \text{" " + INLET CONNECTIONS}$$

2) DIAMOND DRILL + O-RING JOINT;

$$V_{DD} = 0.785 (1.916)^2 (143.625)^*$$

$$= 413.9 \text{ IN}^3 \text{ TEST STRING}$$

$$\approx 414.5 \text{ IN}^3 \quad \text{" " + INLET CONNECTIONS}$$

DETERMINE THE VOLUME RATE LOSS IN A STRING OF LONGYEAR AND A STRING OF DIAMOND DRILL - UNDER VARYING CONDITIONS OF THE JOINT BOXES & PINS BEING EITHER DRY OR WITH LUBRICANT.

1. LONGYEAR

$$V_{05} = 408 \text{ IN}^3$$

$$\frac{P_1 V_1}{T_1} = \frac{P_2 V_2}{T_2} \quad T_1 = T_2$$

$$\therefore P_1 V_1 = P_2 V_2$$

1) FROM TABLE 1

USING AVERAGE OF NUMBERS

a) $98.3 \rightarrow 90$ PSIG $t = 4$ SEC

$$P_1 V_1 = P_2 V_2$$

$$98.3(408) = 90 V_2$$

$$V_2 = 445.6 \text{ IN}^3$$

$$V_2 - V_1 = 445.6 - 408 = 37.6 \text{ IN}^3$$

$$Q = V_{\text{LOSS}}/t$$

$$= \frac{37.6 \text{ IN}^3}{4 \text{ SEC}} = 9.4 \text{ IN}^3/\text{SEC}$$

$$Q = 19.6 \text{ CF/HR} = 0.33 \text{ CFM}$$

$$\frac{\text{IN}^3}{\text{SEC}} \times \frac{1 \text{ CF}}{1728 \text{ IN}^3} \times \frac{60 \text{ SEC}}{1 \text{ MIN}} \\ \times \frac{1 \text{ CF}}{28.8 \text{ MIN}} = 0.0347 \text{ CFM}$$

$$\frac{\text{IN}^3}{\text{SEC}} \times \frac{\text{CF}}{1728 \text{ IN}^3} \times \frac{3600 \text{ SEC}}{\text{HR}} \\ \times 2.083 \text{ CF/HR}$$

b) $90 \rightarrow 80$ PSIG, $t = 7.67$ SEC

$$P_1 V_1 = P_2 V_2$$

$$(90)408 = 80 V_2$$

$$V_2 = 459 \text{ IN}^3$$

$$V_2 - V_1 = 459 - 408 = 51 \text{ IN}^3$$

$$Q = V_{\text{LOSS}}/t = \frac{51 \text{ IN}^3}{7.67 \text{ SEC}} = 6.65 \text{ IN}^3/\text{S}$$

$$Q = 6.65 \text{ CI/S} (2.083)$$

$$= 13.85 \text{ CF/HR} = 0.23 \text{ CFM}$$

c) $80 \rightarrow 70$ PSIG, $t = 9.63$ S

$$P_1 V_1 = P_2 V_2$$

$$80(408) = 70 V_2$$

$$V_2 = 466.3 \text{ IN}^3$$

$$V_2 - V_1 = 466.3 - 408$$

$$\Delta V = 58.3 \text{ IN}^3$$

$$Q = V_{\text{LOSS}}/t = \frac{58.3}{9.63}$$

$$= 6.05 \text{ CI/S} (2.083)$$

$$Q = 12.61 \text{ CF/HR} = 0.22 \text{ CFM}$$

WHC-SD-WM-TRP-251

REV. 0

Page 8f of 10

$$d) 70 \rightarrow 60 \text{ PSIG}, t = 11.0 \text{ s}$$

$$P_1 V_1 = P_2 V_2$$

$$70(408) = 60 V_2$$

$$V_2 = 476 \text{ IN}^3$$

$$V_2 - V_1 = 476 - 408 = 68 \text{ IN}^3$$

$$Q = \frac{V_{\text{loss}}}{t} = \frac{68}{11}$$

$$Q = 6.18 \text{ IN}^3/\text{s} (2.053)$$

$$Q = 12.9 \text{ CF/HR} = 0.22 \text{ CFM}$$

$$e) 60 \rightarrow 50 \text{ PSIG}, t = 13 \text{ s}$$

$$P_1 V_1 = P_2 V_2$$

$$60(408) = 50 V_2$$

$$V_2 = 489.6 \text{ IN}^3$$

$$V_2 - V_1 = 489.6 - 408 = 81.6 \text{ IN}^3$$

$$Q = \frac{V_{\text{loss}}}{t} = \frac{81.6}{13}$$

$$= 6.3 \text{ IN}^3/\text{SEC} (2.053)$$

$$Q = 13.1 \text{ CF/HR} = 0.22 \text{ CFM}$$

$$f) 50 \rightarrow 40 \text{ PSIG}, t = 17 \text{ s}$$

$$P_1 V_1 = P_2 V_2$$

$$50(408) = 40 V_2$$

$$V_2 = 510 \text{ IN}^3$$

$$V_2 - V_1 = 510 - 408 = 102 \text{ IN}^3$$

$$Q = \frac{V_{\text{loss}}}{t} = \frac{102}{17} = 6.0 \text{ IN}^3/\text{SEC} (2.053)$$

$$Q = 12.5 \text{ CF/HR} = 0.21 \text{ CFM}$$

$$g) 40 \rightarrow 30 \text{ PSIG}, t = 20 \text{ s}$$

$$P_1 V_1 = P_2 V_2$$

$$40(408) = 30 V_2$$

$$V_2 = 544 \text{ IN}^3$$

$$V_2 - V_1 = 544 - 408 = 136 \text{ IN}^3$$

$$Q = \frac{V_{\text{loss}}}{t} = \frac{136}{20} = 6.8 \text{ IN}^3/\text{s} (2.053)$$

$$Q = 14.2 \text{ CF/HR} = 0.24 \text{ CFM}$$

h) 30 → 20 PSIG, $t = 29.7$ s

$$P_1 V_1 = P_2 V_2$$

$$30(408) = 20 V_2$$

$$V_2 = 612 \text{ IN}^3$$

$$V_2 - V_1 = 612 - 408 = 204 \text{ IN}^3$$

$$Q = \frac{V}{t} = \frac{204}{29.7} = 6.9 \text{ IN}^3/\text{s} (2.083)$$

$$Q = 14.3 \text{ CF/HR} = 0.24 \text{ CFM}$$

i) 20 → 10 PSIG, $t = 46.3$ s

$$P_1 V_1 = P_2 V_2$$

$$20(408) = 10 V_2$$

$$V_2 = 816 \text{ IN}^3$$

$$V_2 - V_1 = 816 - 408 = 408 \text{ IN}^3$$

$$Q = \frac{V}{t} = \frac{408}{46.3} = 8.81 \text{ IN}^3/\text{s} (2.083)$$

$$Q = 18.4 \text{ CF/HR} = 0.31 \text{ CFM}$$

98.3-90	90-80	80-70	70-60	60-50	50-40	40-30	30-20	20-10	
19.6	13.85	12.61	12.9	13.1	12.5	14.2	14.3	18.4	

 ΔP - PRESSURE DROP (PSIG)

- GRAPH OF TABLE 1

2) FROM TABLE 2

USING AVERAGE NUMBERS

a) 98.3 → 95 PSIG, $t = 527.3$ SEC

$$P_1 V_1 = P_2 V_2$$

$$98.3(408) = 95 V_2$$

$$V_2 = 422.2 \text{ IN}^3$$

$$V_2 - V_1 = 422.2 - 408$$

$$V_2 = 14.2 \text{ IN}^3$$

$$Q = \frac{V}{t} = \frac{14.2}{527.3} = 0.0269 \text{ IN}^3/\text{s} (2.083)$$

$$Q = 0.056 \text{ CF/HR} = 0.001 \text{ CFM}$$

b) 95 → 85 PSIG, $t = 72.7$ s

$$P_1 V_1 = P_2 V_2$$

$$95(408) = 85 V_2$$

$$V_2 = 456 \text{ IN}^3$$

$$V_2 - V_1 = 456 - 408 = 48 \text{ IN}^3$$

$$Q = \frac{V}{t} = \frac{48}{72.7} = 0.6602 \text{ IN}^3/\text{s} (2.083)$$

$$Q = 1.38 \text{ CF/HR} = 0.023 \text{ CFM}$$

- GRAPH OF TABLE 2

WHC-SD-WM-TRP-251

REV. 0

Page 8h of 10

JM

Q
CF/HR

3) FROM TABLE 3

USING AVERAGE NUMBERS

$$99 \rightarrow 98.2 \text{ PSIG}, t = 600 \text{ s}$$

$$P_1 V_1 = P_2 V_2$$

$$99(408) = 98.2 V_2$$

$$V_2 = 411.3 \text{ IN}^3$$

$$V_2 - V_1 = 411.3 - 408 = 3.3 \text{ IN}^3$$

$$Q = V/t = \frac{3.3}{600} = 0.0055 \text{ IN}^3/\text{s} (2.053)$$

$$Q = 0.0115 \text{ CF/HR} = 0.0002 \text{ CFM}$$

- PUT DOT ON GRAPH -

- NO PRESSURE LOSS -

2. DIAMOND DRILL

$$V_1 = 411.6 \text{ IN}^3 \text{ (NO O-RING JT)}$$

$$V_2 = 414.5 \text{ IN}^3 \text{ (W/O-RING JT)}$$

1) FROM TABLE 4

USING AVERAGE NUMBERS

$$98 \rightarrow 92.9 \text{ PSIG}, t = 600 \text{ s}$$

$$P_1 V_1 = P_2 V_2$$

$$98(411.6) = 92.9 V_2$$

$$V_2 = 434.2 \text{ IN}^3$$

$$V_2 - V_1 = 434.2 - 411.6 = 22.6 \text{ IN}^3$$

$$Q = V/t = \frac{22.6}{600} = 0.0377 \text{ IN}^3/\text{s} (2.083)$$

$$Q = 0.078 \text{ CF/HR} = 0.0013 \text{ CFM}$$

- PUT DOT ON GRAPH -

2) FROM TABLE 5

$$a) V = 414.5 \text{ IN}^3$$

$$98.3 \rightarrow 90 \text{ PSIG}, t = 270 \text{ s}$$

$$P_1 V_1 = P_2 V_2$$

$$98.3(414.5) = 90 V_2$$

$$V_2 = 452.7 \text{ IN}^3$$

$$V_2 - V_1 = 452.7 - 414.5 = 38.2 \text{ IN}^3$$

$$Q = V/t = \frac{38.2}{270} = 0.142 \text{ IN}^3/\text{s} (2.083)$$

$$Q = 0.29 \text{ CF/HR} = 0.0048 \text{ CFM}$$

1/13/96

CALCULATIONS (CONT.)

51
51b) 90 - 81 PSIG, $t = 252.7$ s

$$P_1 V_1 = P_2 V_2$$

$$90(414.5) = 81 V_2$$

$$V_2 = 460.56 \text{ IN}^3$$

$$V_2 - V_1 = 460.56 - 414.5 = 46.1 \text{ IN}^3$$

$$Q = V/t = 46.1 / 252.7 = 0.182 \text{ IN}^3/\text{s} (2.083)$$

$$Q = 0.38 \text{ CF/HR} = 0.0063 \text{ CFM}$$

c) 81 \rightarrow 70 1/4 PSIG, $t = 77.7$ s

$$P_1 V_1 = P_2 V_2$$

$$81(414.5) = 70.25 V_2$$

$$V_2 = 478 \text{ IN}^3$$

$$V_2 - V_1 = 478 - 414.5 = 63.4 \text{ IN}^3$$

$$Q = V/t = \frac{63.4}{77.7} = 0.816 \text{ IN}^3/\text{s} (2.083)$$

$$Q = 1.7 \text{ CF/HR} = 0.028 \text{ CFM}$$

— PLOT GRAPH FOR TABLE 5

3. FROM TABLE 6

98 \rightarrow 96.7 PSIG, $t = 600$ s

$$P_1 V_1 = P_2 V_2$$

$$98(414.5) = 96.7 V_2$$

$$V_2 = 420.1 \text{ IN}^3$$

$$V_2 - V_1 = 420.1 - 414.5 = 5.6 \text{ IN}^3$$

$$Q = V/t = 5.6 / 600 = 0.0093 \text{ IN}^3/\text{SEC} (2.083)$$

$$Q = 0.019 \text{ CF/HR} = 0.0003 \text{ CFM}$$

— PLOT POINT ON GRAPH —

— NO PRESSURE LOSS —

4. FROM TABLE 7

97.5 \rightarrow 97.25 PSIG, $t = 600$ s

$$P_1 V_1 = P_2 V_2$$

$$97.5(414.6) = 97.25 V_2$$

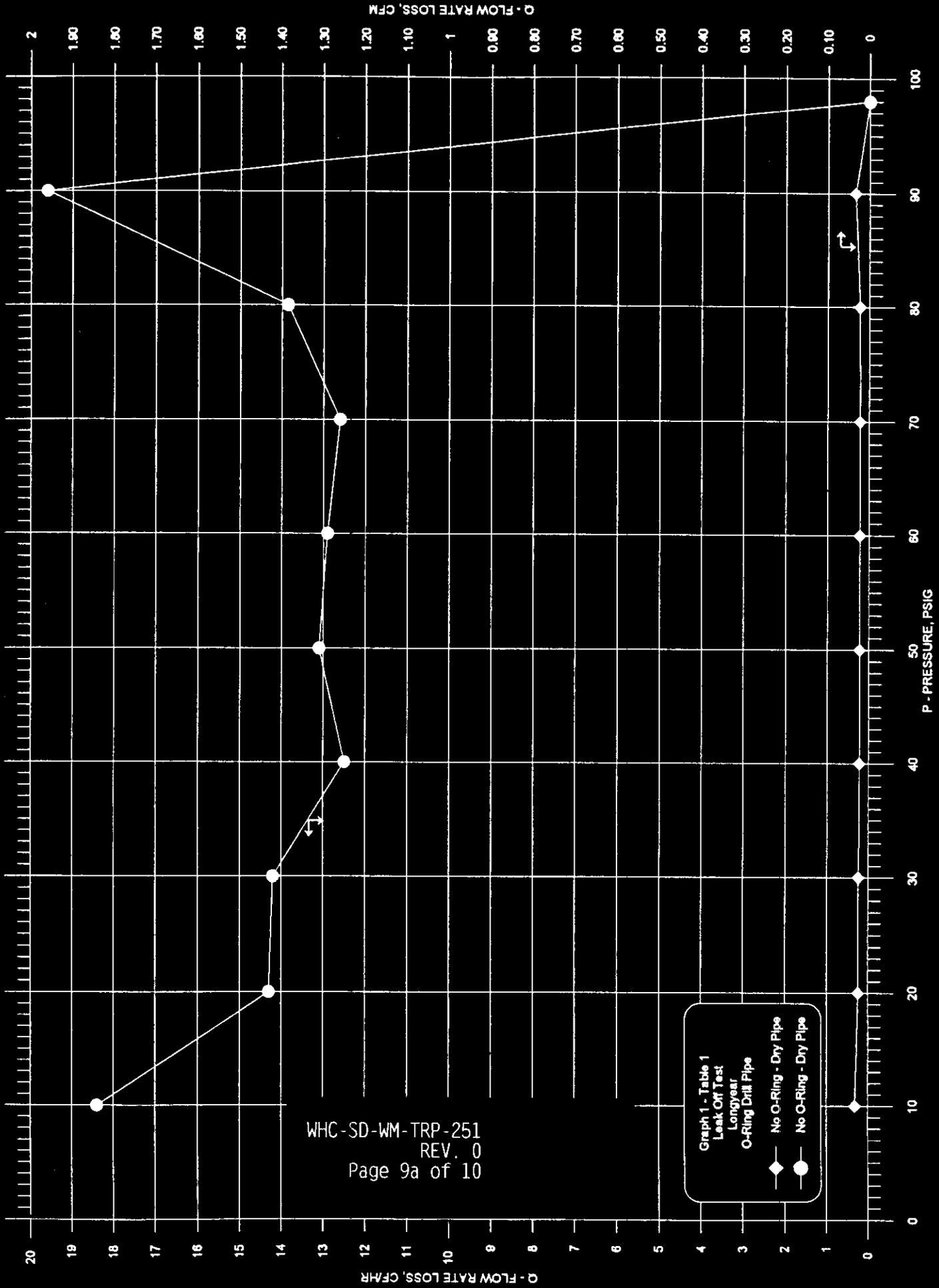
$$V_2 = 410.5 \text{ IN}^3$$

— NO PRESSURE LOSS —

PLOT POINT ON GRAPH

6.0 Appendix B - Data Graphics

Arrangement 1



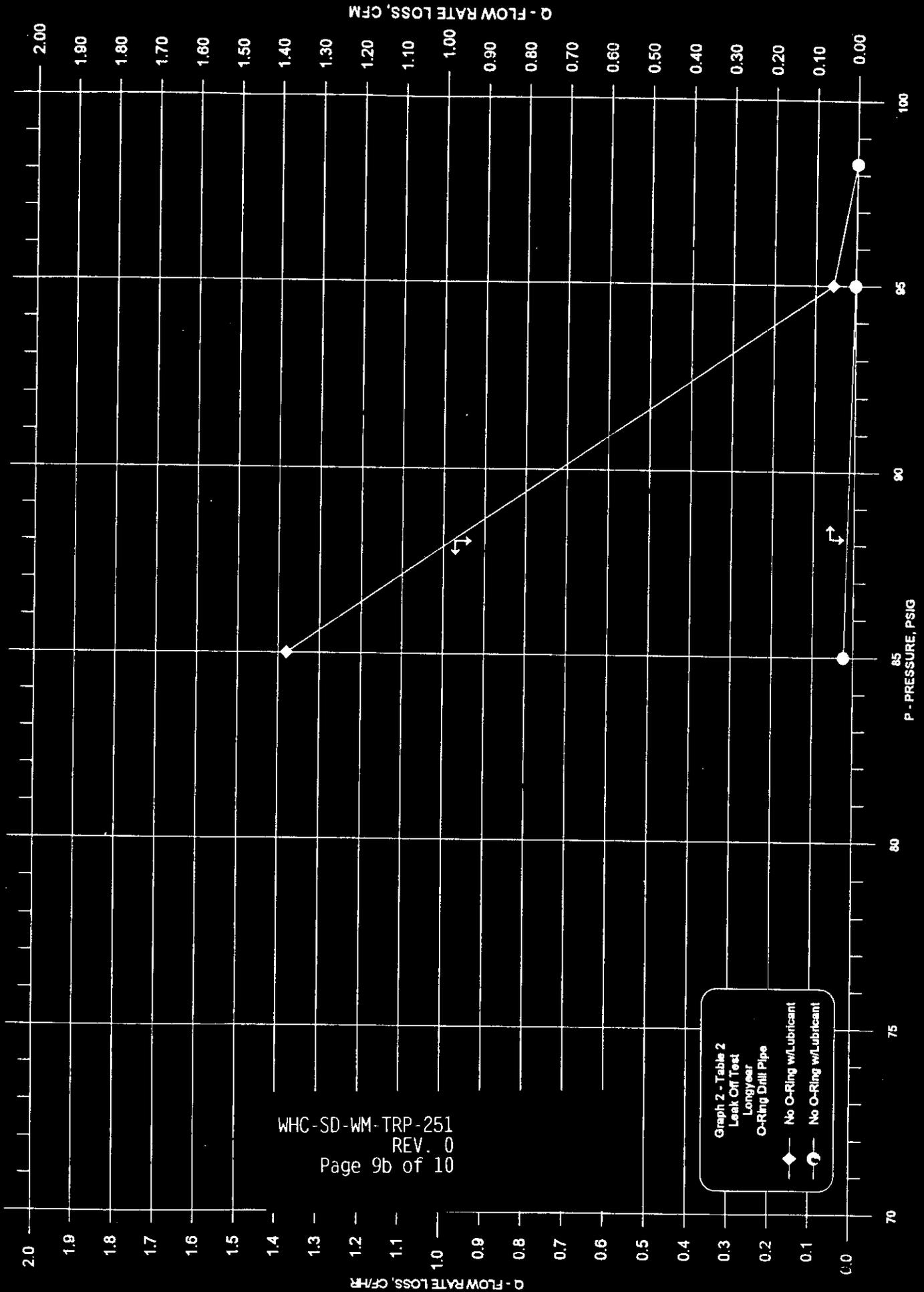
WHC-SD-WM-TRP-251
 REV. 0
 Page 9a of 10

Graph 1 - Table 1
 Leak Off Test
 Longyear
 O-Ring Drill Pipe
 ◆ No O-Ring - Dry Pipe
 ● No O-Ring - Dry Pipe

Q - FLOW RATE LOSS, CFM

P - PRESSURE, PSIG

Arrangement 2

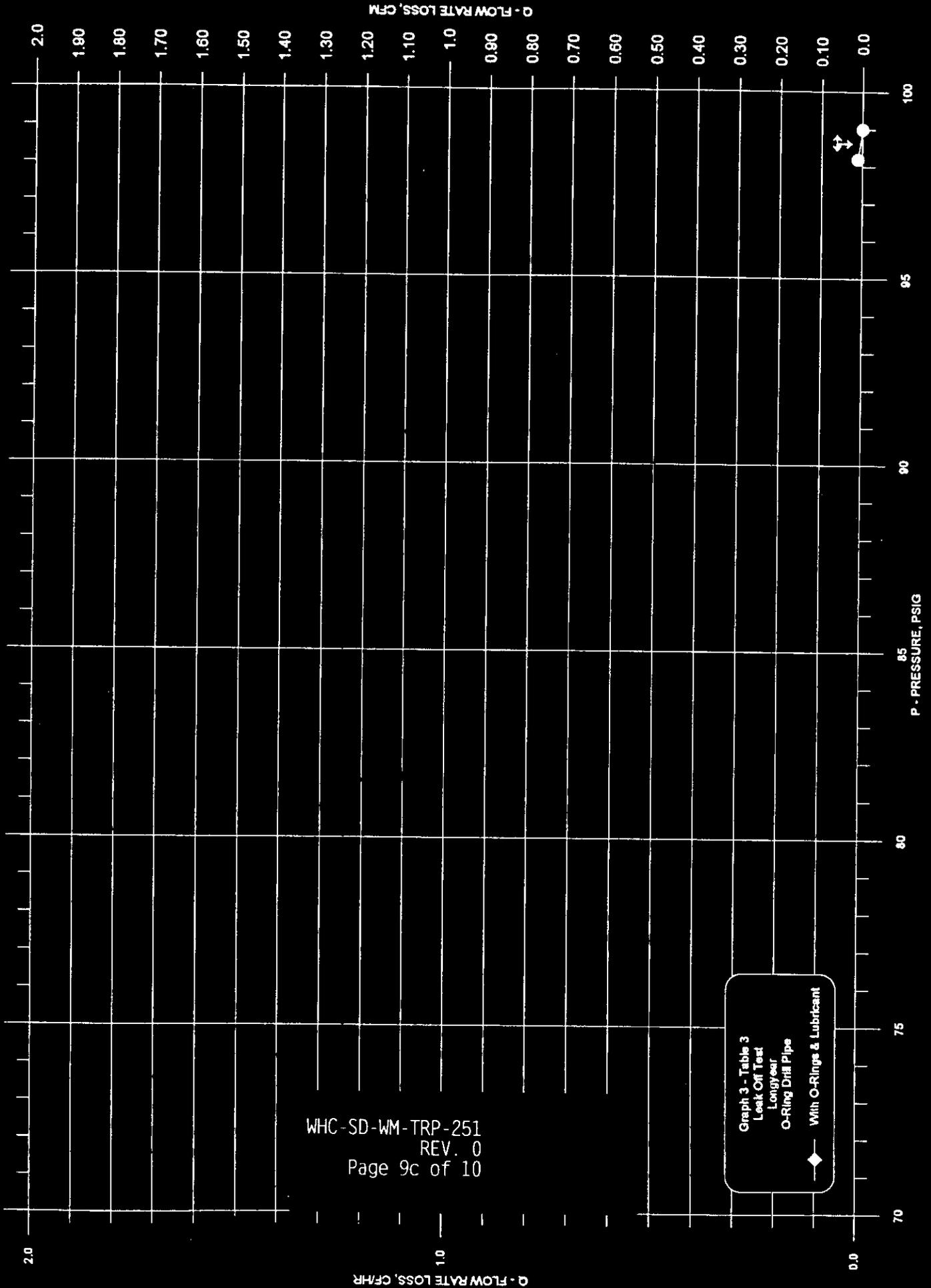


WHC-SD-WM-TRP-251
 REV. 0
 Page 9b of 10

Graph 2 - Table 2
 Leak Off Test
 Longyear
 O-Ring Drill Pipe

◆ No O-Ring w/Lubricant
 ● No O-Ring w/Lubricant

Arrangement 3

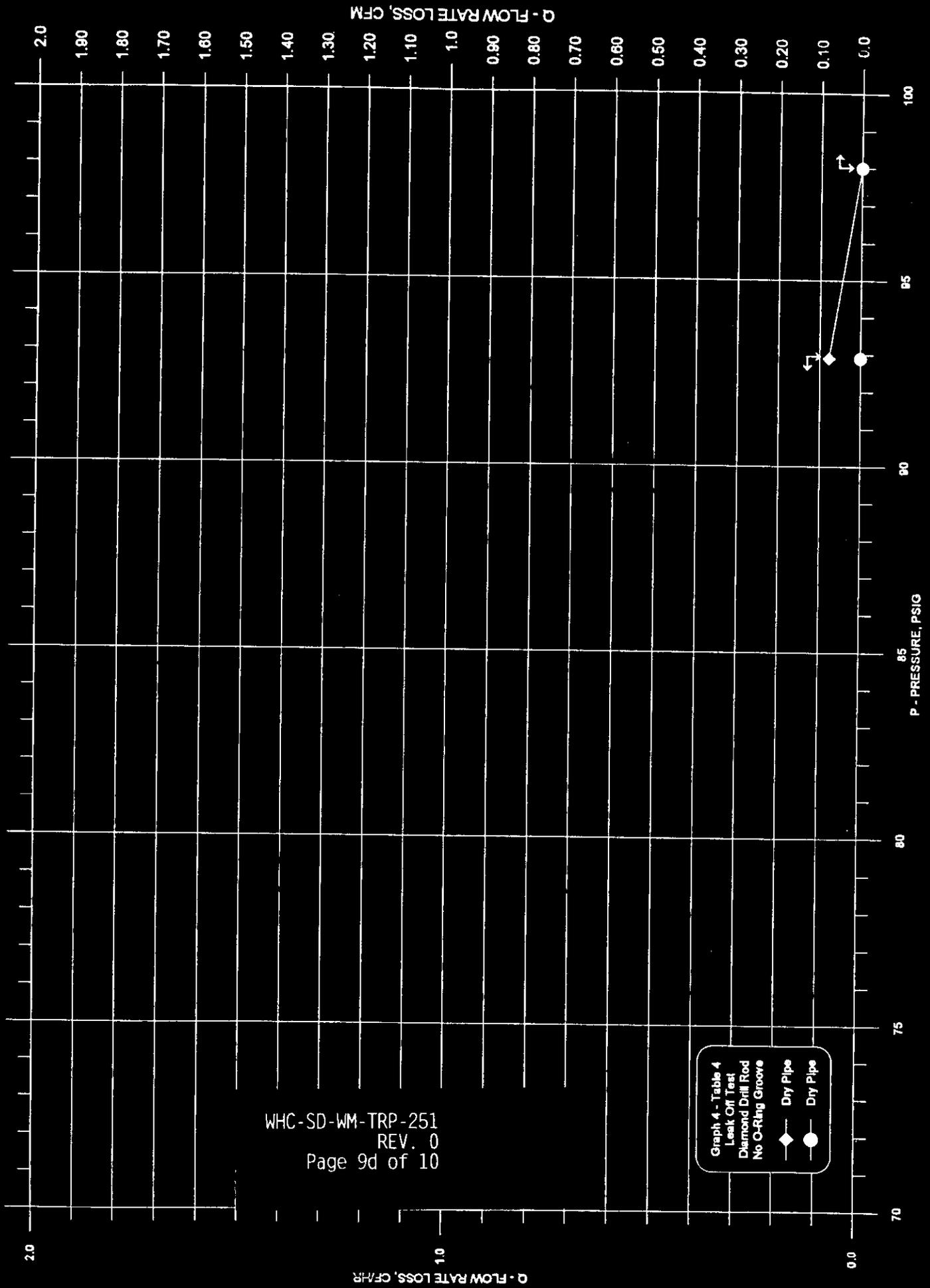


WHC-SD-WM-TRP-251
REV. 0
Page 9c of 10

Graph 3 - Table 3
Leak Off Test
Longyear
O-Ring Drill Pipe
◆ With O-Rings & Lubricant

Arrangement 4

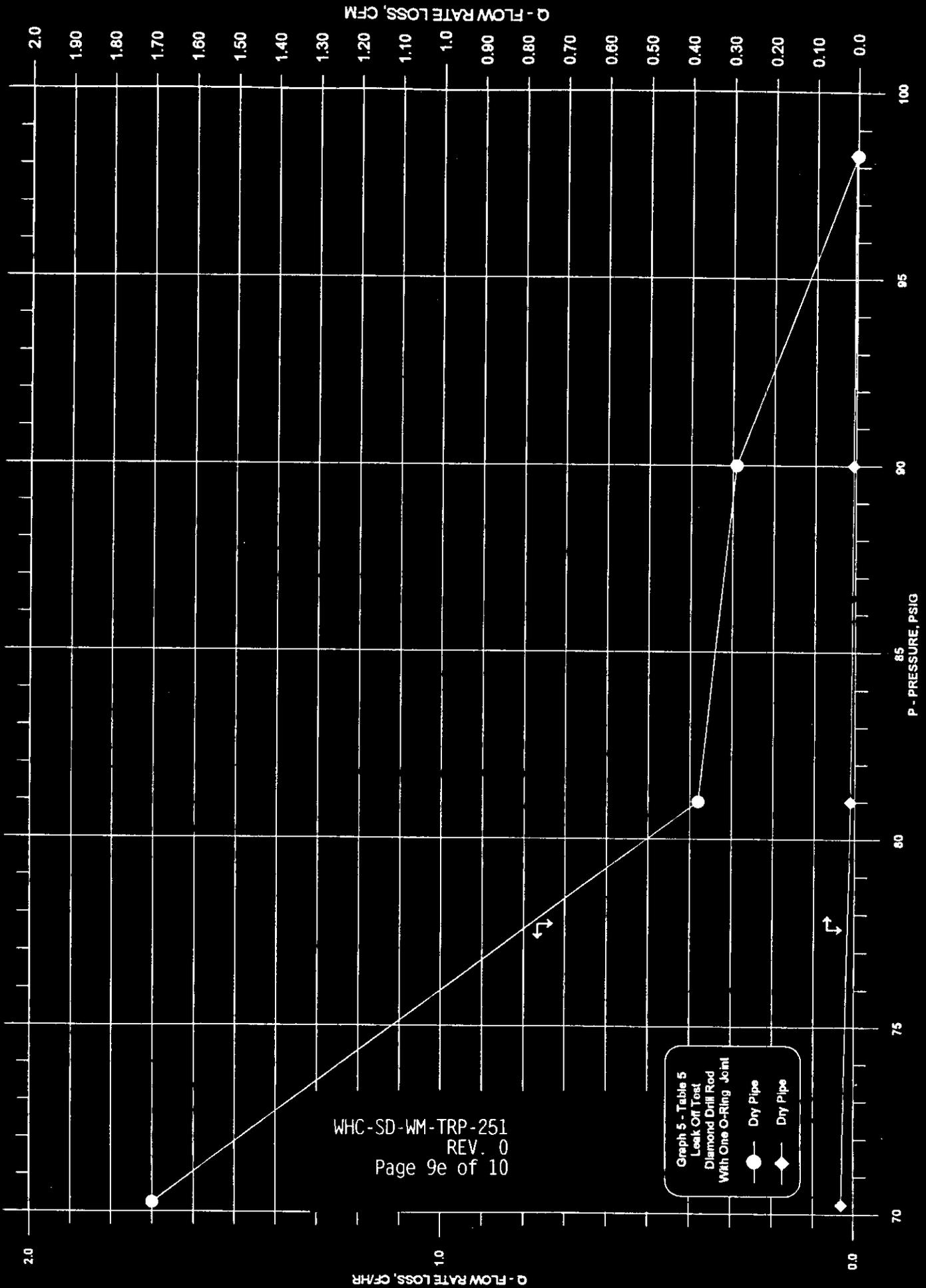
WHC-SD-WM-TRP-251
REV. 0
Page 9d of 10



Graph 4 - Table 4
Leak Off Test
Diamond Drill Rod
No O-Ring Groove

- ◆ Dry Pipe
- Dry Pipe

Arrangement 5



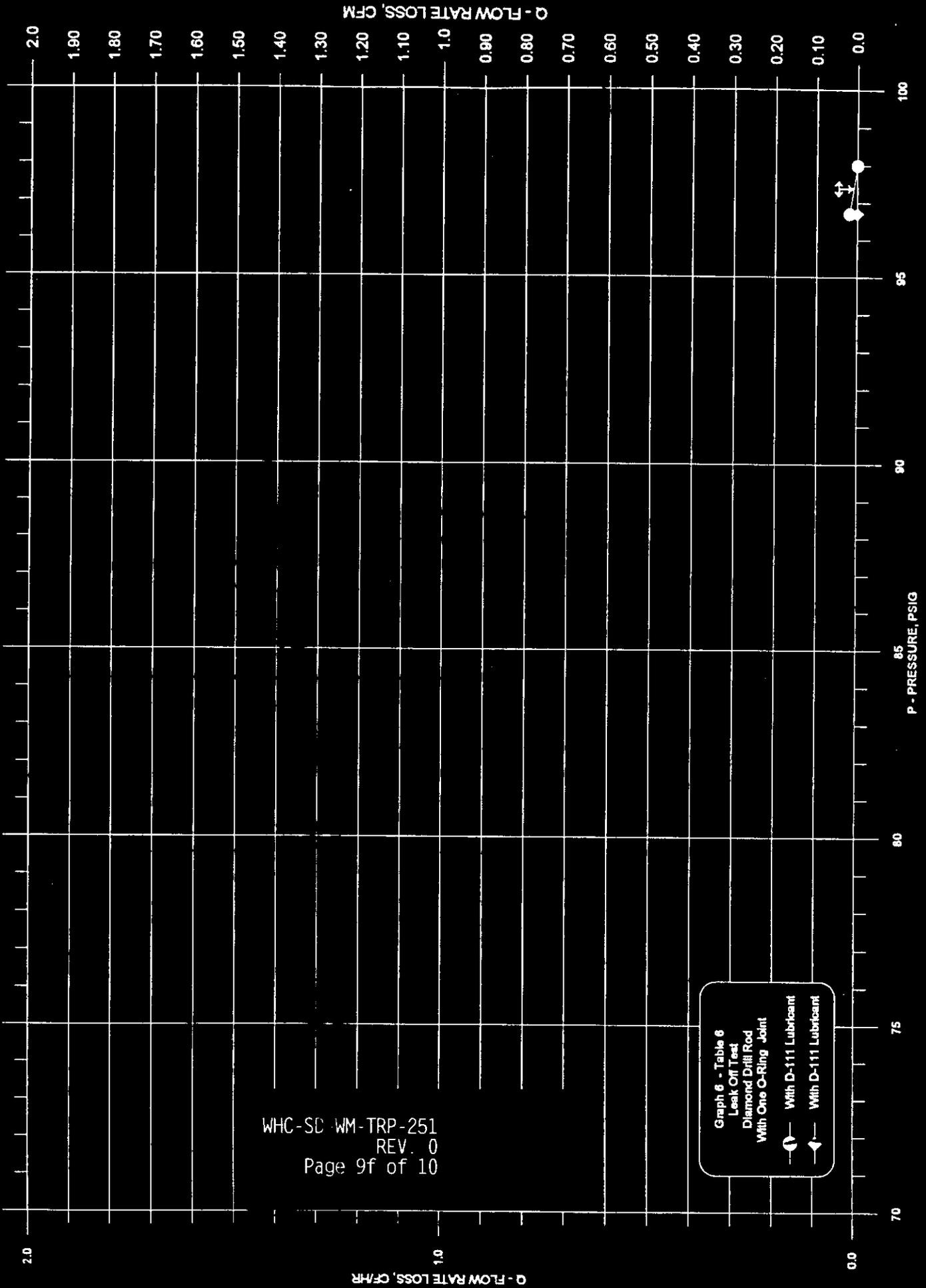
WHC-SD-WM-TRP-251
REV. 0
Page 9e of 10

Q - FLOW RATE LOSS, CF/HR

Q - FLOW RATE LOSS, CFM

P - PRESSURE, PSIG

Arrangement 6

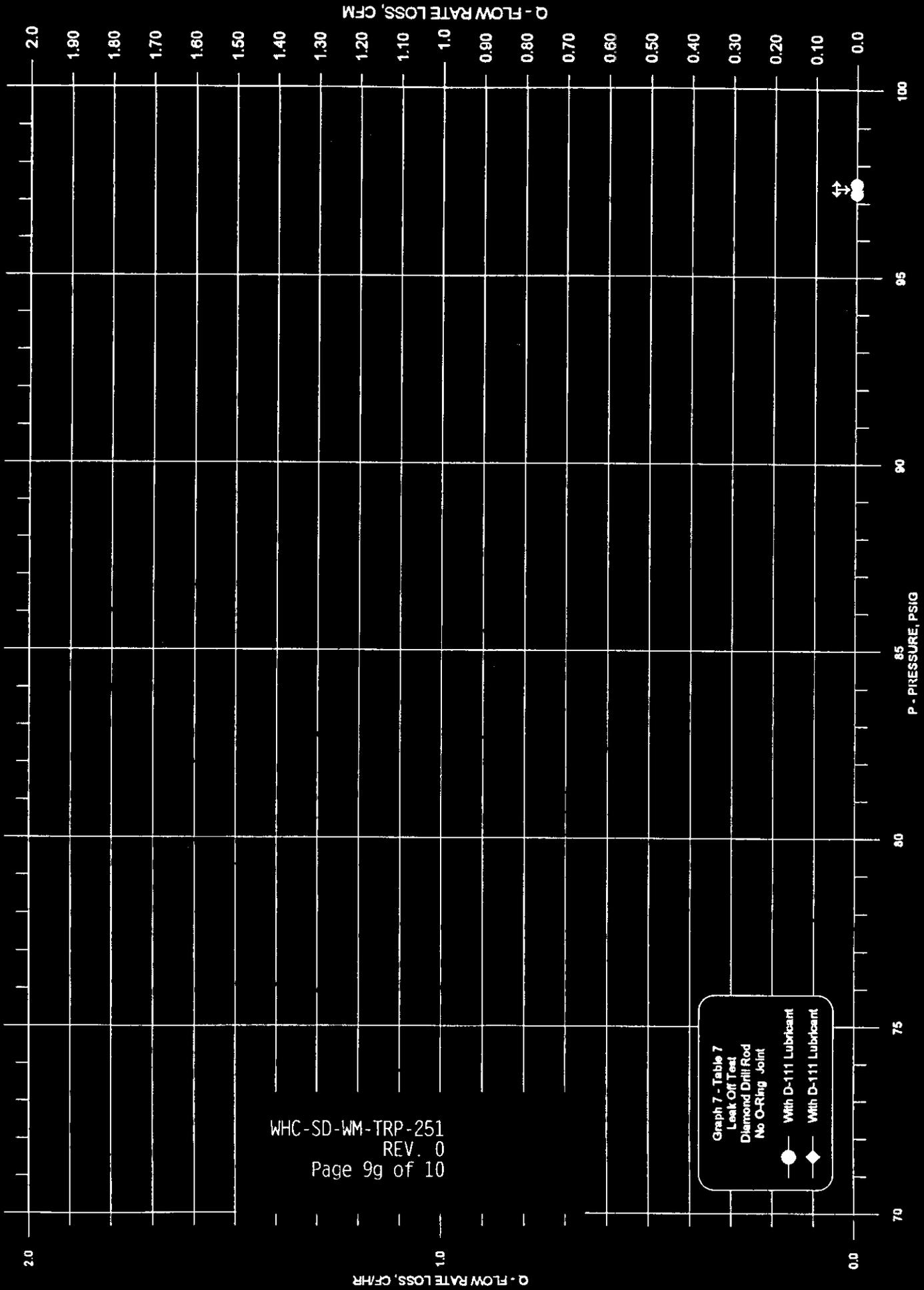


WHC-SD-WM-TRP-251
 REV. 0
 Page 9f of 10

Graph 6 - Table 6
 Leak Off Test
 Diamond Drill Rod
 With One O-Ring Joint

 With D-111 Lubricant
 With D-111 Lubricant

Arrangement 7



WHC-SD-WM-TRP-251
 REV. 0
 Page 9g of 10

Graph 7 - Table 7
 Leak Off Test
 Diamond Drill Rod
 No O-Ring Joint

● With D-111 Lubricant
 ◆ With D-111 Lubricant

WHC-SD-WM-TRP-251

REV. 0

Page 10 of 10

7.0 Appendix C - Photos

Bench test assembly and air hose attached to inlet manifold. Steel cage in background to which the Longyear drill rod was tied down.

Bench test assembly in 305 Building test pit.

Bench clamp stand and Diamond Drill rod test string. Inlet connections and pressure gauge.